10-59 A composite wall consists of several horizontal and vertical layers. The left and right surfaces of the wall are maintained at uniform temperatures. The rate of heat transfer through the wall, the interface temperatures, and the temperature drop across the section F are to be determined.

Assumptions 1 Heat transfer is steady since there is no indication of change with time. 2 Heat transfer through the wall is one-dimensional. 3 Thermal conductivities are constant. 4 Thermal contact resistances at the interfaces are disregarded.

Properties The thermal conductivities are given to be $k_A = k_F = 2$, $k_B = 8$, $k_C = 20$, $k_D = 15$, $k_E = 35$ W/m·°C.

Analysis (a) The representative surface area is $A = 0.12 \times 1 = 0.12 \,\mathrm{m}^2$. The thermal resistance network and the individual thermal resistances are

$$R_{1} = R_{A} = \left(\frac{L}{kA}\right)_{A} = \frac{0.01 \,\mathrm{m}}{(2 \,\mathrm{W/m} \cdot {}^{\circ}\mathrm{C})(0.12 \,\mathrm{m}^{2})} = 0.04 \,{}^{\circ}\mathrm{C/W}$$

$$R_{2} = R_{4} = R_{C} = \left(\frac{L}{kA}\right)_{C} = \frac{0.05 \,\mathrm{m}}{(20 \,\mathrm{W/m} \cdot {}^{\circ}\mathrm{C})(0.04 \,\mathrm{m}^{2})} = 0.16 \,{}^{\circ}\mathrm{C/W}$$

$$R_{3} = R_{B} = \left(\frac{L}{kA}\right)_{B} = \frac{0.05 \,\mathrm{m}}{(8 \,\mathrm{W/m} \cdot {}^{\circ}\mathrm{C})(0.04 \,\mathrm{m}^{2})} = 0.16 \,{}^{\circ}\mathrm{C/W}$$

$$R_{5} = R_{D} = \left(\frac{L}{kA}\right)_{D} = \frac{0.1 \,\mathrm{m}}{(15 \,\mathrm{W/m} \cdot {}^{\circ}\mathrm{C})(0.06 \,\mathrm{m}^{2})} = 0.11 \,{}^{\circ}\mathrm{C/W}$$

$$R_{6} = R_{E} = \left(\frac{L}{kA}\right)_{E} = \frac{0.1 \,\mathrm{m}}{(35 \,\mathrm{W/m} \cdot {}^{\circ}\mathrm{C})(0.06 \,\mathrm{m}^{2})} = 0.05 \,{}^{\circ}\mathrm{C/W}$$

$$R_{7} = R_{F} = \left(\frac{L}{kA}\right)_{F} = \frac{0.06 \,\mathrm{m}}{(2 \,\mathrm{W/m} \cdot {}^{\circ}\mathrm{C})(0.12 \,\mathrm{m}^{2})} = 0.25 \,{}^{\circ}\mathrm{C/W}$$

$$\frac{1}{R_{mid,1}} = \frac{1}{R_{2}} + \frac{1}{R_{3}} + \frac{1}{R_{4}} = \frac{1}{0.06} + \frac{1}{0.16} + \frac{1}{0.06} \longrightarrow R_{mid,1} = 0.025 \,{}^{\circ}\mathrm{C/W}$$

$$\frac{1}{R_{mid,2}} = \frac{1}{R_{5}} + \frac{1}{R_{6}} = \frac{1}{0.11} + \frac{1}{0.05} \longrightarrow R_{mid,2} = 0.034 \,{}^{\circ}\mathrm{C/W}$$

$$R_{total} = R_{1} + R_{mid,1} + R_{mid,2} + R_{7} = 0.04 + 0.025 + 0.034 + 0.25 = 0.349 \,{}^{\circ}\mathrm{C/W}$$

$$\dot{Q} = \frac{T_{col} - T_{col}}{R_{total}} = \frac{(300 - 100) \,{}^{\circ}\mathrm{C}}{0.349 \,{}^{\circ}\mathrm{C/W}} = 572 \,\mathrm{W} \text{ (for a } 0.12 \,\mathrm{m} \times 1 \,\mathrm{m} \text{ section)}$$

Then steady rate of heat transfer through entire wall becomes

$$\dot{Q}_{total} = (572 \text{ W}) \frac{(5 \text{ m})(8 \text{ m})}{0.12 \text{ m}^2} = 1.91 \times 10^5 \text{ W}$$

(b) The total thermal resistance between left surface and the point where the sections B, D, and E meet is $R_{total} = R_1 + R_{mid,1} = 0.04 + 0.025 = 0.065$ °C/W

Then the temperature at the point where the sections B, D, and E meet becomes

$$\dot{Q} = \frac{T_1 - T}{R_{total}} \longrightarrow T = T_1 - \dot{Q}R_{total} = 300^{\circ}\text{C} - (572 \text{ W})(0.065 ^{\circ}\text{C/W}) = 263^{\circ}\text{C}$$

(c) The temperature drop across the section F can be determined from

$$\dot{Q} = \frac{\Delta T}{R_F} \to \Delta T = \dot{Q}R_F = (572 \text{ W})(0.25 \text{ °C/W}) = 143 \text{ °C}$$