

11-36 Tomatoes are placed into cold water to cool them. The heat transfer coefficient and the amount of heat transfer are to be determined.

Assumptions **1** The tomatoes are spherical in shape. **2** Heat conduction in the tomatoes is one-dimensional because of symmetry about the midpoint. **3** The thermal properties of the tomatoes are constant. **4** The heat transfer coefficient is constant and uniform over the entire surface. **5** The Fourier number is $\tau > 0.2$ so that the one-term approximate solutions (or the transient temperature charts) are applicable (this assumption will be verified).

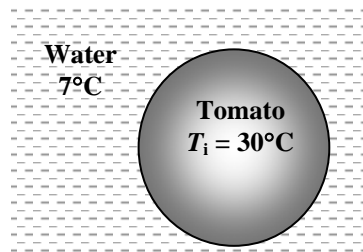
Properties The properties of the tomatoes are given to be $k = 0.59 \text{ W/m}\cdot^\circ\text{C}$, $\alpha = 0.141 \times 10^{-6} \text{ m}^2/\text{s}$, $\rho = 999 \text{ kg/m}^3$ and $c_p = 3.99 \text{ kJ/kg}\cdot^\circ\text{C}$.

Analysis The Fourier number is

$$\tau = \frac{\alpha t}{r_o^2} = \frac{(0.141 \times 10^{-6} \text{ m}^2/\text{s})(2 \times 3600 \text{ s})}{(0.04 \text{ m})^2} = 0.635$$

which is greater than 0.2. Therefore one-term solution is applicable. The ratio of the dimensionless temperatures at the surface and center of the tomatoes are

$$\frac{\theta_{s,\text{sph}}}{\theta_{0,\text{sph}}} = \frac{\frac{T_s - T_\infty}{T_i - T_\infty}}{\frac{T_0 - T_\infty}{T_i - T_\infty}} = \frac{T_s - T_\infty}{T_0 - T_\infty} = \frac{A_1 e^{-\lambda_1^2 \tau} \frac{\sin(\lambda_1)}{\lambda_1}}{A_1 e^{-\lambda_1^2 \tau}} = \frac{\sin(\lambda_1)}{\lambda_1}$$



Substituting,

$$\frac{7.1 - 7}{10 - 7} = \frac{\sin(\lambda_1)}{\lambda_1} \longrightarrow \lambda_1 = 3.0401$$

From Table 11-2, the corresponding Biot number and the heat transfer coefficient are

$$\text{Bi} = 31.1$$

$$\text{Bi} = \frac{hr_o}{k} \longrightarrow h = \frac{k\text{Bi}}{r_o} = \frac{(0.59 \text{ W/m}\cdot^\circ\text{C})(31.1)}{(0.04 \text{ m})} = \mathbf{459 \text{ W/m}^2\cdot^\circ\text{C}}$$

The maximum amount of heat transfer is

$$m = 8\rho V = 8\rho\pi D^3 / 6 = 8(999 \text{ kg/m}^3)[\pi(0.08 \text{ m})^3 / 6] = 2.143 \text{ kg}$$

$$Q_{\max} = mc_p [T_i - T_\infty] = (2.143 \text{ kg})(3.99 \text{ kJ/kg}\cdot^\circ\text{C})(30 - 7)^\circ\text{C} = 196.6 \text{ kJ}$$

Then the actual amount of heat transfer becomes

$$\left(\frac{Q}{Q_{\max}} \right)_{\text{cyl}} = 1 - 3 \left(\frac{T_0 - T_\infty}{T_i - T_\infty} \right) \frac{\sin \lambda_1 - \lambda_1 \cos \lambda_1}{\lambda_1^3} = 1 - 3 \left(\frac{10 - 7}{30 - 7} \right) \frac{\sin(3.0401) - (3.0401) \cos(3.0401)}{(3.0401)^3} = 0.9565$$

$$Q = 0.9565 Q_{\max}$$

$$Q = 0.9565(196.6 \text{ kJ}) = \mathbf{188 \text{ kJ}}$$