**8-86** Liquid water is heated in a chamber by mixing it with superheated steam. For a specified mixing temperature, the mass flow rate of the steam and the rate of exergy destruction are to be determined.

**Assumptions 1** This is a steady-flow process since there is no change with time. **2** Kinetic and potential energy changes are negligible. **3** There are no work interactions.

**Properties** Noting that  $T < T_{\text{sat }@200 \text{ kPa}} = 120.23^{\circ}\text{C}$ , the cold water and the exit mixture streams exist as a compressed liquid, which can be approximated as a saturated liquid at the given temperature. From Tables A-4 through A-6,

$$\begin{array}{l} P_1 = 200 \; \mathrm{kPa} \\ T_1 = 15^{\circ}\mathrm{C} \end{array} \right\} \begin{array}{l} h_1 \cong h_{f@15^{\circ}\mathrm{C}} = 62.98 \; \mathrm{kJ/kg} \\ S_1 \cong s_{f@15^{\circ}\mathrm{C}} = 0.22447 \; \mathrm{kJ/kg \cdot K} \end{array}$$
 
$$\begin{array}{l} P_2 = 200 \; \mathrm{kPa} \\ T_2 = 200^{\circ}\mathrm{C} \end{array} \right\} \begin{array}{l} h_2 = 2870.4 \; \mathrm{kJ/kg} \\ S_2 = 7.5081 \; \mathrm{kJ/kg \cdot K} \end{array}$$
 
$$\begin{array}{l} R_3 \cong h_{f@80^{\circ}\mathrm{C}} = 335.02 \; \mathrm{kJ/kg} \\ T_3 = 80^{\circ}\mathrm{C} \end{array} \right\} \begin{array}{l} h_3 \cong h_{f@80^{\circ}\mathrm{C}} = 335.02 \; \mathrm{kJ/kg} \\ S_3 \cong s_{f@80^{\circ}\mathrm{C}} = 1.0756 \; \mathrm{kJ/kg \cdot K} \end{array}$$

000 kJ/min

15°C
4 kg/s
MIXING
CHAMBER
200 kPa

200°C
3

*Analysis* (a) We take the mixing chamber as the system, which is a control volume. The mass and energy balances for this steady-flow system can be expressed in the rate form as

Mass balance: 
$$\dot{m}_{\rm in} - \dot{m}_{\rm out} = \Delta \dot{m}_{\rm system}$$
  $\stackrel{\Leftrightarrow 0}{\Leftrightarrow} ({\rm steady}) = 0 \longrightarrow \dot{m}_1 + \dot{m}_2 = \dot{m}_3$ 

Energy balance:

Combining the two relations gives  $\dot{Q}_{\text{out}} = \dot{m}_1 h_1 + \dot{m}_2 h_2 - (\dot{m}_1 + \dot{m}_2) h_3 = \dot{m}_1 (h_1 - h_3) + \dot{m}_2 (h_2 - h_3)$ 

Solving for  $\dot{m}_2$  and substituting, the mass flow rate of the superheated steam is determined to be

$$\dot{m}_2 = \frac{\dot{Q}_{\text{out}} - \dot{m}_1 (h_1 - h_3)}{h_2 - h_3} = \frac{(600/60 \text{ kJ/s}) - (4 \text{ kg/s})(62.98 - 335.02) \text{kJ/kg}}{(2870.4 - 335.02) \text{kJ/kg}} = \textbf{0.429 kg/s}$$

Also, 
$$\dot{m}_3 = \dot{m}_1 + \dot{m}_2 = 4 + 0.429 = 4.429 \,\text{kg/s}$$

(b) The exergy destroyed during a process can be determined from an exergy balance or directly from its definition  $X_{\text{destroyed}} = T_0 S_{\text{gen}}$  where the entropy generation  $S_{\text{gen}}$  is determined from an entropy balance on an *extended* system that includes the mixing chamber and its immediate surroundings. It gives

$$\frac{\dot{S}_{\text{in}} - \dot{S}_{\text{out}}}{\text{Rate of net entropy transfer}} + \underbrace{\dot{S}_{\text{gen}}}_{\text{Rate of entropy}} = \underbrace{\Delta \dot{S}_{\text{system}}}_{\text{Rate of change}} = 0$$

$$\frac{\dot{R}_{\text{ate of net entropy}}}{\dot{R}_{\text{ate of entropy}}} = 0$$

$$\dot{m}_{1}s_{1} + \dot{m}_{2}s_{2} - \dot{m}_{3}s_{3} - \frac{\dot{Q}_{\text{out}}}{T_{\text{b,surr}}} + \dot{S}_{\text{gen}} = 0 \quad \Rightarrow \dot{S}_{\text{gen}} = \dot{m}_{3}s_{3} - \dot{m}_{1}s_{1} - \dot{m}_{2}s_{2} + \frac{\dot{Q}_{\text{out}}}{T_{0}}$$

Substituting, the exergy destruction is determined to be

$$\dot{X}_{\text{destroyed}} = T_0 \dot{S}_{\text{gen}} = T_0 \left( \dot{m}_3 s_3 - \dot{m}_2 s_2 - \dot{m}_1 s_1 + \frac{\dot{Q}_{out}}{T_{b,surr}} \right) 
= (298 \text{ K})(4.429 \times 1.0756 - 0.429 \times 7.5081 - 4 \times 0.22447 + 10 / 298) \text{kW/K} 
= 202 \text{ kW}$$

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