**9-39** An ideal Otto cycle with air as the working fluid has a compression ratio of 9.5. The highest pressure and temperature in the cycle, the amount of heat transferred, the thermal efficiency, and the mean effective pressure are to be determined.

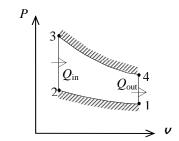
Assumptions 1 The air-standard assumptions are applicable. 2 Kinetic and potential energy changes are negligible. 3 Air is an ideal gas with constant specific heats.

**Properties** The properties of air at room temperature are  $c_p = 1.005 \text{ kJ/kg·K}$ ,  $c_v = 0.718 \text{ kJ/kg·K}$ , R = 0.287 kJ/kg·K, and k = 1.4 (Table A-2).

Analysis (a) Process 1-2: isentropic compression.

$$T_2 = T_1 \left(\frac{\mathbf{v}_1}{\mathbf{v}_2}\right)^{k-1} = (308 \text{ K})(9.5)^{0.4} = 757.9 \text{ K}$$

$$\frac{P_2 \mathbf{v}_2}{T_2} = \frac{P_1 \mathbf{v}_1}{T_1} \longrightarrow P_2 = \frac{\mathbf{v}_1}{\mathbf{v}_2} \frac{T_2}{T_1} P_1 = (9.5) \left(\frac{757.9 \text{ K}}{308 \text{ K}}\right) (100 \text{ kPa}) = 2338 \text{ kPa}$$



Process 3-4: isentropic expansion.

$$T_3 = T_4 \left(\frac{v_4}{v_3}\right)^{k-1} = (800 \text{ K})(9.5)^{0.4} = 1969 \text{ K}$$

Process 2-3: v = constant heat addition.

$$\frac{P_3 \mathbf{v}_3}{T_3} = \frac{P_2 \mathbf{v}_2}{T_2} \longrightarrow P_3 = \frac{T_3}{T_2} P_2 = \left(\frac{1969 \text{ K}}{757.9 \text{ K}}\right) (2338 \text{ kPa}) = \mathbf{6072 \text{ kPa}}$$

(b) 
$$m = \frac{P_1 \mathbf{V}_1}{RT_1} = \frac{(100 \text{ kPa})(0.0006 \text{ m}^3)}{(0.287 \text{ kPa} \cdot \text{m}^3/\text{kg} \cdot \text{K})(308 \text{ K})} = 6.788 \times 10^{-4} \text{kg}$$

$$Q_{\rm in} = m(u_3 - u_2) = mc_{\nu}(T_3 - T_2) = (6.788 \times 10^{-4} \,\mathrm{kg})(0.718 \,\mathrm{kJ/kg \cdot K})(1969 - 757.9) \,\mathrm{K} = \mathbf{0.590 \,kJ}$$

(c) Process 4-1:  $\mathbf{v}$  = constant heat rejection.

$$Q_{\text{out}} = m(u_4 - u_1) = mc_v (T_4 - T_1) = -(6.788 \times 10^{-4} \text{ kg})(0.718 \text{ kJ/kg} \cdot \text{K})(800 - 308) \text{K} = 0.240 \text{ kJ}$$

$$W_{\text{net}} = Q_{\text{in}} - Q_{\text{out}} = 0.590 - 0.240 = 0.350 \text{ kJ}$$

$$\eta_{\text{th}} = \frac{W_{\text{net,out}}}{Q_{\text{in}}} = \frac{0.350 \text{ kJ}}{0.590 \text{ kJ}} = \mathbf{59.4\%}$$

(d) 
$$V_{\min} = V_2 = \frac{V_{\max}}{r}$$

$$MEP = \frac{W_{\text{net,out}}}{V_1 - V_2} = \frac{W_{\text{net,out}}}{V_1(1 - 1/r)} = \frac{0.350 \text{ kJ}}{(0.0006 \text{ m}^3)(1 - 1/9.5)} \left(\frac{\text{kPa} \cdot \text{m}^3}{\text{kJ}}\right) = 652 \text{ kPa}$$