

9-94 A simple Brayton cycle with air as the working fluid operates between the specified temperature and pressure limits. The effects of non-isentropic compressor and turbine on the back-work ratio is to be compared.

Assumptions 1 Steady operating conditions exist. 2 The air-standard assumptions are applicable. 3 Kinetic and potential energy changes are negligible. 4 Air is an ideal gas with constant specific heats.

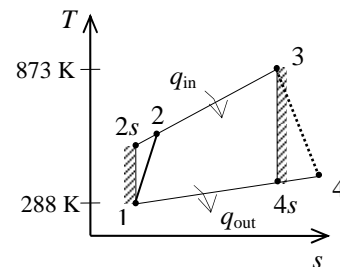
Properties The properties of air at room temperature are $c_p = 1.005 \text{ kJ/kg}\cdot\text{K}$ and $k = 1.4$ (Table A-2a).

Analysis For the compression process,

$$T_{2s} = T_1 \left(\frac{P_2}{P_1} \right)^{(k-1)/k} = (288 \text{ K})(12)^{0.4/1.4} = 585.8 \text{ K}$$

$$\eta_C = \frac{h_{2s} - h_1}{h_2 - h_1} = \frac{c_p(T_{2s} - T_1)}{c_p(T_2 - T_1)} \longrightarrow T_2 = T_1 + \frac{T_{2s} - T_1}{\eta_C}$$

$$= 288 + \frac{585.8 - 288}{0.90} = 618.9 \text{ K}$$



For the expansion process,

$$T_{4s} = T_3 \left(\frac{P_4}{P_3} \right)^{(k-1)/k} = (873 \text{ K}) \left(\frac{1}{12} \right)^{0.4/1.4} = 429.2 \text{ K}$$

$$\eta_T = \frac{h_3 - h_4}{h_3 - h_{4s}} = \frac{c_p(T_3 - T_4)}{c_p(T_3 - T_{4s})} \longrightarrow T_4 = T_3 - \eta_T(T_3 - T_{4s})$$

$$= 873 - (0.90)(873 - 429.2)$$

$$= 473.6 \text{ K}$$

The isentropic and actual work of compressor and turbine are

$$W_{\text{Comp},s} = c_p(T_{2s} - T_1) = (1.005 \text{ kJ/kg}\cdot\text{K})(585.8 - 288) \text{ K} = 299.3 \text{ kJ/kg}$$

$$W_{\text{Comp}} = c_p(T_2 - T_1) = (1.005 \text{ kJ/kg}\cdot\text{K})(618.9 - 288) \text{ K} = 332.6 \text{ kJ/kg}$$

$$W_{\text{Turb},s} = c_p(T_3 - T_{4s}) = (1.005 \text{ kJ/kg}\cdot\text{K})(873 - 429.2) \text{ K} = 446.0 \text{ kJ/kg}$$

$$W_{\text{Turb}} = c_p(T_3 - T_4) = (1.005 \text{ kJ/kg}\cdot\text{K})(873 - 473.6) \text{ K} = 401.4 \text{ kJ/kg}$$

The back work ratio for 90% efficient compressor and isentropic turbine case is

$$r_{\text{bw}} = \frac{W_{\text{Comp}}}{W_{\text{Turb},s}} = \frac{332.6 \text{ kJ/kg}}{446.0 \text{ kJ/kg}} = \mathbf{0.7457}$$

The back work ratio for 90% efficient turbine and isentropic compressor case is

$$r_{\text{bw}} = \frac{W_{\text{Comp},s}}{W_{\text{Turb}}} = \frac{299.3 \text{ kJ/kg}}{401.4 \text{ kJ/kg}} = \mathbf{0.7456}$$

The two results are almost identical.