

# *Reacting Gas Mixtures*



**Reading**  
14-1 → 14-7

**Problems**  
14-16, 14-22, 14-42, 14-48, 14-68,  
14-77, 14-91, 14-93

## **Introduction**

- thermodynamic analysis of reactive mixtures is primarily an extension of the principles we have learned thus far
- it is necessary to modify the methods used to calculate specific enthalpy, internal energy and entropy

## **Definitions**

### ***Combustion Process:***

- a fuel made up of hydrocarbons is said to have burned completely if:
  - all the carbon present in the fuel is burned to carbon dioxide
  - all the hydrogen is burned to water
- if the conditions are not fulfilled the combustion process is incomplete

### ***Combustion Reactions:***

reactants → products

or

fuel + oxidizer → products

- in all cases the mass is conserved

mass of products = mass of reactants

### ***Fuels:***

- fuel is simply a combustible substance
- hydrocarbon fuels exist as liquids, gases and solids
  - liquids → gasoline - octane,  $C_8H_{18}$

- gases → methane,  $CH_4$
- solids → coal

### Combustion Air:

- oxygen is required in every combustion reaction
- in most combustion reactions air provides the needed oxygen
- dry air is considered to be

$\left. \begin{array}{l} 21\% \text{ oxygen} \\ 79\% \text{ nitrogen} \end{array} \right\}$  on a molar basis

$$molar\ ratio = \frac{n_{N_2}}{n_{O_2}} = \frac{0.79}{0.21} = 3.76$$

1 mole of air can then be written as  $[0.21 O_2 + 0.79 N_2]$

For convenience, we typically refer to air as  $[O_2 + 3.76 N_2]$  which is actually 4.76 moles of air.

**Note:** From the Amagat model, we know that a mixture at a known  $T$  and  $P$  (as is the case with combustion reactions)

$$\frac{n_i}{n} = \frac{V_i}{V}$$

Therefore, by expressing a mixture in terms of the number of moles we are also expressing it in terms of a volume fraction.

- nitrogen does not undergo a chemical reaction in combustion since it is inert

### Air-Fuel Ratio:

$$\frac{mass\ of\ air}{mass\ of\ fuel} = \frac{moles\ of\ air \times \tilde{M}_{air}}{moles\ of\ fuel \times \tilde{M}_{fuel}}$$

$$AF = \bar{AF} \left( \frac{\tilde{M}_{air}}{\tilde{M}_{fuel}} \right)$$

where:

$AF$  – air fuel ratio on a mass basis

$\bar{AF}$  – air fuel ratio on a molar basis

$\tilde{M}_{air}$  = 28.97 kg/kmole

### ***Theoretical or Stoichiometric Air:***

- the minimum amount of air that supplies sufficient oxygen for complete combustion of all carbon and hydrogen in the fuel - referred to as stoichiometric, 100% stoichiometric or theoretical
- no free oxygen would appear in the products
- greater than stoichiometric leads to free oxygen in the products
- less than stoichiometric and  $C$ ,  $CO$ ,  $OH$ ,  $H_2$  will appear in the products since there is not enough oxygen to form water or carbon dioxide (the actual proportions will depend on the temperature and the pressure)
- normally the amount of air supplied is given as a percentage of the theoretical value  
i.e.  $150\% = 1.5 \times$  the theoretical air  
- referred to as 20% excess air, 120% stoichiometric

### ***Equivalence Ratio:***

- defined as

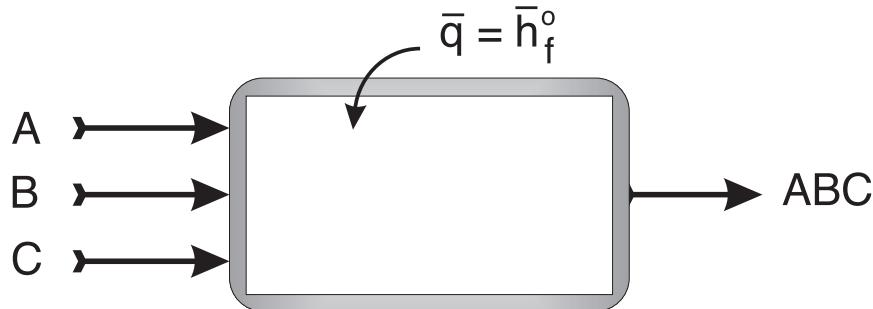
$$\text{equivalence ratio} = \frac{AF_{actual}}{AF_{theoretical}}$$

- if the equivalence ratio is:
  - $> 1$   $\rightarrow$  lean mixture (excess air)
  - $< 1$   $\rightarrow$  rich mixture (not enough air)

# Conservation of Energy for Reacting Systems

## Enthalpy of Formation

- previous calculations involving enthalpy were all based on differences and the reference used to determine enthalpy did not matter
- when chemical reactions occur, reactants disappear and products are formed  
→ differences cannot be calculated for all substances involved
- it is necessary to establish a common base to account for differences in composition
- the enthalpy datum for reacting systems is set to zero at standard temperature and pressure
  - $T_{ref} = 25^\circ C \rightarrow 298 K$
  - $P_{ref} = 1 atm$
- $h = 0$  assigned to elements in their most stable form i.e.  $O_2, N_2, C, etc.$
- **Enthalpy of Formation:** *the energy released or absorbed when a compound is formed from its stable elements at STP*



where  $\bar{h}_f^o$  is the enthalpy of formation.

Taking an energy balance over the combustion chamber shown above, we obtain

$$\underbrace{a \bar{h}_A^o + b \bar{h}_B^o + c \bar{h}_C^o}_{generally=0} + \bar{h}_f^o \longrightarrow \bar{h}_{ABC}^o$$

The left side of the equation is typically zero because  $h = 0$  for elements in their stable form. The sign of  $\bar{h}_f^o$  indicates the direction of heat flow; +ve is endothermic and -ve is exothermic.

## Effects of Non-Standard Temperature

$$\bar{h}(T, P) = \bar{h}_f^o + \underbrace{(\bar{h}_{T,P} - \bar{h}_{T=25 \text{ } ^\circ C, P=1 \text{ atm}}^o)}_{\Delta \bar{h} \text{ at known temperatures}}$$

where

$\bar{h}_f^o$  is the heat resulting from a chemical change at  $T = 25 \text{ } ^\circ C$  and  $P = 1 \text{ atm}$

$\Delta \bar{h}$  is the heat resulting from a change in temperature (sensible heat) with respect to the reference temperature,  $T_{ref} = 25 \text{ } ^\circ C$

# Enthalpy of Combustion

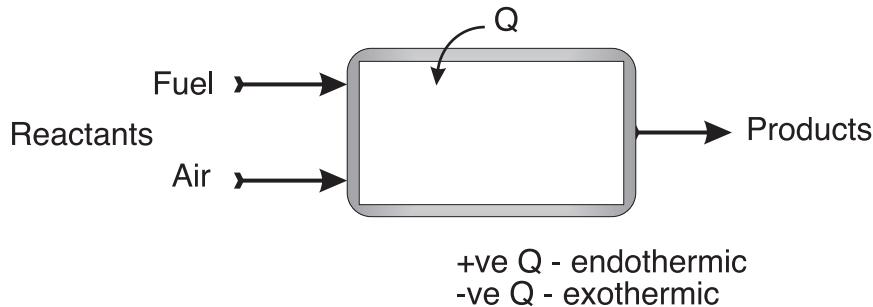
- while the enthalpy of formation is related to elemental reactants → resulting in a single compound; the **enthalpy of combustion** is related to *fuel + oxidizer* as the reactants
- Enthalpy of Combustion:** *the difference between the enthalpy of the products and the enthalpy of the reactants where complete combustion occurs at a given temperature and pressure*

$$Q = \sum(mh)_P - \sum(mh)_R = \underbrace{H_P(T_P) - H_R(T_R)}_{H_{RP}}$$

$$Q = \sum(n\bar{h})_P - \sum(n\bar{h})_R = \underbrace{\bar{H}_P(T_P) - \bar{H}_R(T_R)}_{\bar{H}_{RP}}$$

where

$$\bar{h}_c = H_{RP}/\text{kmole of fuel}$$



with:

$$+ve Q \Rightarrow \text{endothermic}$$

$$-ve Q \Rightarrow \text{exothermic}$$

- when enthalpy of formation data are available for all products and reactants the above equation can be used
- otherwise a calorimeter must be used to measure the enthalpy of combustion

## Heating Value

- the heating value of a fuel is a positive value equal to the magnitude of the enthalpy of combustion when products are returned to the state of the reactants
- two values are used
  - **HHV**: higher heating value - obtained when all the water formed by combustion is a liquid at the reference temperature
  - **LHV**: lower heating value - obtained when all the water formed by combustion is a vapour as an ideal gas in the mixture of the products
- the HHV exceeds the LHV by the energy required to vaporize the liquid formed

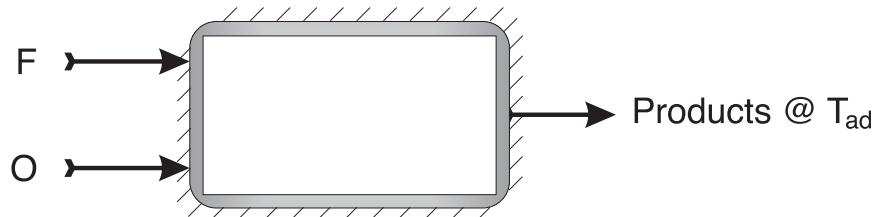
$$\begin{aligned} HHV &= LHV + \frac{(m \cdot h_{fg})_{H_2O}}{kmole \text{ of fuel}} \\ &= LHV + (\tilde{M} \cdot h_{fg})_{H_2O} \cdot \frac{n_{H_2O}}{n_{fuel}} \end{aligned}$$

where

$$h_{fg}(25^\circ C) = 2,442.3 \text{ kJ/kg}$$

$$\tilde{M}_{H_2O} = 18.015 \text{ kg/kmole}$$

## Adiabatic Flame Temperature



- if the system is perfectly insulated it cannot dispose of the LHV and the LHV goes into heating the products above the reference temperature

- under adiabatic conditions, the maximum temperature attained by the products when combustion is complete is called the adiabatic flame or adiabatic combustion temperature

$$H_P(T_{ad}) = H_R(T_R)$$

$$\sum_P n_P (\bar{h}_f^o + \underbrace{\bar{h} - \bar{h}^o}_{\Delta \bar{h}})_P = \sum_R n_R (\bar{h}_f^o + \underbrace{\bar{h} - \bar{h}^o}_{\Delta \bar{h}})_R$$

We need to collect terms based on what we know or can readily calculate and what we do not know, i.e. terms that are a function of  $T_{ad}$ .

$$\begin{aligned} \sum_P \underbrace{n_P (\bar{h})_P}_{\substack{\text{sensible heat} \\ \text{function of } T_{ad}}} &= \sum_R \underbrace{n_R (\bar{h} - \bar{h}^o)_R - \left( -\sum_P n_P (\bar{h}^o)_P \right)}_{\substack{\text{sensible heat} \\ \text{function of } T_R \text{ or } T_{ref}}} \\ &+ \underbrace{\sum_R n_R (\bar{h}_f^o)_R - \sum_P n_P (\bar{h}_f^o)_P}_{\substack{\text{chemical heat} \\ \text{function of } T_R \text{ or } T_{ref}}} \end{aligned}$$

**Step 1:** Calculate the right hand side based on known values of  $T_R$  and  $T_{ref}$ .

**Step 2:** Calculate the left hand side based on a guessed value of  $T_{ad}$ .

**Step 3:** Repeat Step 2, until LHS = RHS.

## Dew Point

- since water is formed when hydrocarbon fuels are burned, the mole fraction of water vapour in the form of gaseous products can be significant
- if the gaseous products of combustion are cooled at constant mixture pressure the dew point temperature is reached when water vapour begins to condense
- since corrosion of duct work, mufflers etc. can occur, the knowledge of dew point temperature is important

## Evaluation of Entropy for Reacting Systems

The 2nd law entropy equation can be written as

$$\underbrace{S_{in} - S_{out}}_{\text{due to heat \& mass transfer}} + \underbrace{S_{gen}}_{\text{generation}} = \underbrace{\Delta S_{system}}_{\text{change in entropy}}$$

For a closed system, such as a combustion process, the entropy balance on the system can be written as

$$\sum \frac{Q_i}{T_i} + S_{gen} = S_P - S_R$$

- a common datum must be used to assign entropy values for each substance involved in the reaction
- an entropy of 0 for pure crystalline substances is obtained at absolute zero
- the entropy relative to this datum is called absolute entropy
- absolute entropy at 1 atm and temperature  $T$  is denoted as  $s^\circ(T)$  or  $\bar{s}^\circ(T)$  for a per unit mass or per mole basis
- while  $\bar{h}$  was only a function of temperature for ideal gases, we must account for the effects of both  $T$  and  $P$  in entropy

- the entropy at any value of  $T$  and  $P$  can be calculated as

$$\bar{s}(T, P) = \underbrace{\bar{s}^o(T)}_{\text{tables}} - \mathcal{R} \ln \left( \frac{P_i}{P_{ref}} \right)$$

where

$$P_{ref} = 1 \text{ atm}$$

$$P_i = \text{partial pressure of } i\text{'th component}$$

$$\mathcal{R} = 8.31434 \text{ kJ/kmole} \cdot \text{K}$$

- the partial pressure  $P_i$  can also be written as

$$P_i = Y_i P$$

and

$$\bar{s}(T, P_i) = \bar{s}_i^o(T) - \mathcal{R} \ln \left( \frac{Y_i P}{P_{ref}} \right)$$

where  $P$  is the mixture pressure and  $Y_i$  is the mole fraction of the  $i\text{'th}$  component.